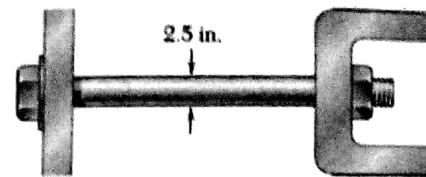




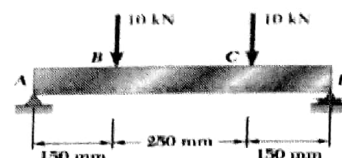
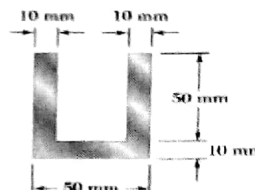
Question 1: [12 Marks]

The change in diameter of a large steel bolt is carefully measured as the nut is tightened. Knowing that $E = 29 \times 10^6$ psi and $\nu = 0.30$, determine the internal force in the bolt, if the diameter is observed to decrease by 0.5×10^{-3} in.



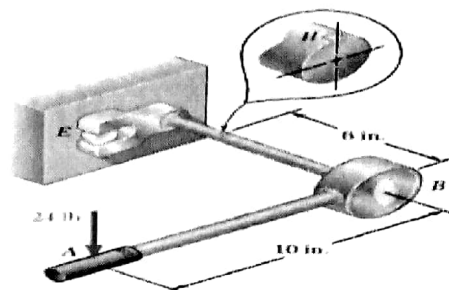
Question 2: [13 Marks]

Two vertical forces are applied to a beam of the cross section shown. Determine the maximum tensile and compressive stresses in portion BC of the beam.



Question 3: [15 Marks]

A mechanic uses a crowfoot wrench to loosen a bolt at E . Knowing that the mechanic applies a vertical 24-lb force at A , determine the principal stresses and the maximum shearing stress at point H located as shown on top of the $3/4\text{-in.}$ diameter shaft. Using analytical and graphical methods.



Question 4: [10 Marks]

1. Principle plan is a plan on which the shear stress is

- (a) Zero (b) Maximum (c) Minimum

2. If the radius of a wire stretched by a load is doubled, it's Young's modulus will be

- (a) Doubled (b) halved (c) Remain the same

3. Two shafts A and B are made of the same material. The shaft A is solid and has diameter D . the shaft B is hollow with outer diameter D and inner diameter $D/2$. The strength of the hollow shaft in torsion is as that of the solid shaft.

- (a) 1/16 (b) 3/4 (c) 15/16

4. A steel bar of 5 mm diameter is heated from 15°C to 40°C and it is free to expand. The bar will induce stress.

- (a) Tensile Thermal (b) Compressive Thermal (c) No

5. The ratio of lateral strain to axial strain is known as

- (a) Elasticity Modulus (b) Material Strength (c) Poisson's Ratio

Best Wishes

Associate Prof. Dr. Noha Fouda

Question 1: [12 Marks]

$$\delta_y = -0.5 \times 10^{-3} \text{ in.}; y_0 = d = 2.5 \text{ in.}$$

$$\varepsilon_y = \frac{\delta_y}{y_0} = -\frac{0.5 \times 10^{-3}}{2.5} = -0.2 \times 10^{-3}$$

$$\nu = -\frac{\varepsilon_y}{\varepsilon_x} \quad \therefore \varepsilon_x = -\frac{\varepsilon_y}{\nu} = \frac{0.2 \times 10^{-3}}{0.3} = 0.66667 \times 10^{-3}$$

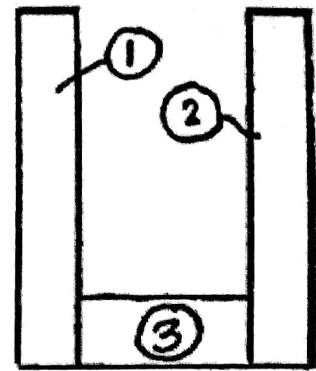
$$\sigma_x = E\varepsilon_x = (29 \times 10^6)(0.66667 \times 10^{-3}) = 19.334 \times 10^3 \text{ psi}$$

$$A = \frac{\pi}{4}d^2 = \frac{\pi}{4}(2.5)^2 = 4.9087 \text{ in}^2$$

$$F = \sigma_x A = (19.334 \times 10^3)(4.9087) = 94.902 \times 10^3 \text{ lb}$$

Question 2: [13 Marks]

	A, mm	\bar{y}_0, mm	$A\bar{y}_0, mm^3$
(1)	600	30	18×10^3
(2)	600	30	18×10^3
(3)	300	5	1.5×10^3
	1500		37.5

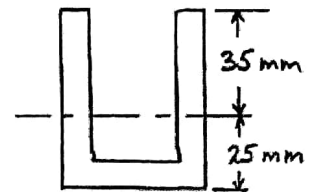


$$\bar{Y}_0 = \frac{37.5 \times 10^3}{1500} = 25 \text{ mm}$$

neutral axis lies 25 mm above the base.

$$I_1 = I_2 = \frac{1}{12} (10)(60)^3 + (600)(5)^2 = 195 \times 10^3 \text{ mm}^4$$

$$I_3 = \frac{1}{12} (300)(10)^3 + (300)(20)^2 = 145 \times 10^3 \text{ mm}^4$$



$$I = I_1 + I_2 + I_3 = 535 \text{ mm}^4$$

$$y_{top} = 35 \text{ mm} = 0.035 \text{ m}; \quad y_{bottom} = -25 \text{ mm} = -0.025 \text{ m}$$

$$a = 150 \text{ mm} = 0.150 \text{ m} \quad P = 10 \times 10^3 \text{ N}$$

$$M = Pa = (10 \times 10^3)(0.150) = 1.5 \times 10^3 \text{ N.m}$$

$$\sigma_{top} = -\frac{My_{top}}{I} = -\frac{(1.5 \times 10^3)(0.035)}{535 \times 10^{-9}} = -98.13 \times 10^6 \text{ Pa (Comp.)}$$

$$\sigma_{bottom} = -\frac{My_{bottom}}{I} = -\frac{(1.5 \times 10^3)(0.025)}{535 \times 10^{-9}} = 70.09 \times 10^6 \text{ Pa (Ten.)}$$

Question 3: [15 Marks]

$$V = 24 \text{ lb}; \quad M = (24)(6) = 144 \text{ lb.in.}; \quad T = (24)(10) = 240 \text{ lb.in.}$$

$$d = 0.75 \text{ in.}; \quad c = \frac{1}{2}d = 0.375 \text{ in.}$$

$$J = \frac{\pi}{2}c^4 = 0.031063 \text{ in}^4; \quad I = \frac{1}{2}J = 0.015532 \text{ in}^4$$

$$\text{Torsion: } \tau = \frac{Tc}{J} = \frac{(24)(0.375)}{0.031063} = 2.897 \times 10^3 \text{ psi} = 2.897 \text{ Ksi.}$$

$$\text{Bending: } \sigma = \frac{Mc}{I} = \frac{(144)(0.375)}{0.015532} = 3.477 \times 10^3 \text{ psi} = 3.477 \text{ Ksi}$$

Transverse shear: At point H stress due to transverse shear is zero.

$$\text{Resultant stresses: } \sigma_x = 3.477 \text{ Ksi}; \quad \sigma_y = 0; \quad \tau_{xy} = 2.897 \text{ Ksi}$$

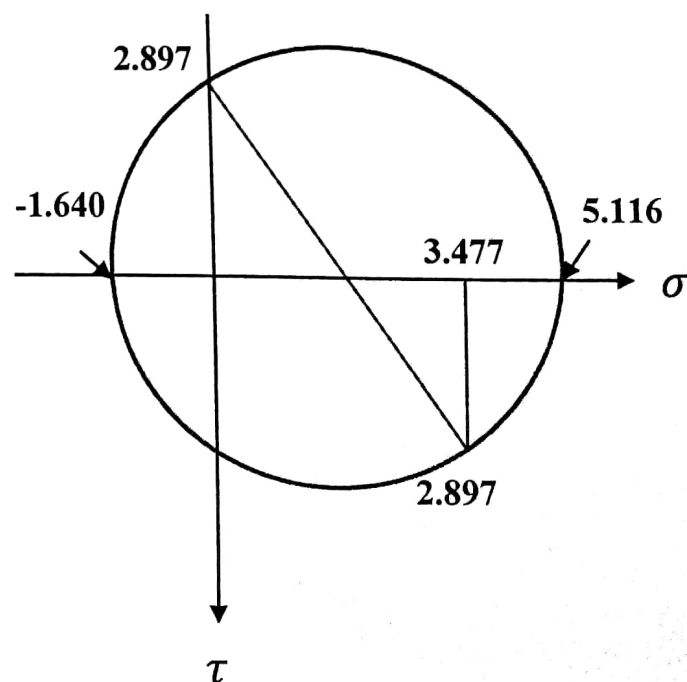
$$\sigma_{avg.} = \frac{1}{2}(\sigma_x + \sigma_y) = 1.738$$

$$R = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} = \sqrt{1.738^2 + 2.897^2} = 3.378 \text{ Ksi}$$

$$\sigma_a = \sigma_{Avg.} + R = 5.116 \text{ Ksi}$$

$$\sigma_b = \sigma_{Avg.} - R = -1.640 \text{ Ksi}$$

$$\tau_{max,min} = R = 3.378 \text{ Ksi}$$



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