

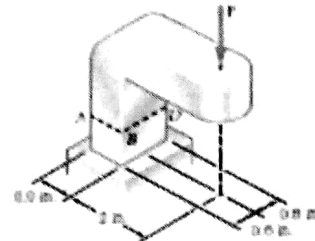
Mid Term Exam  
Strength of materials PDE181  
Time: 1 hrs



Please, solve all the following problems with clear net sketch and force analysis diagrams

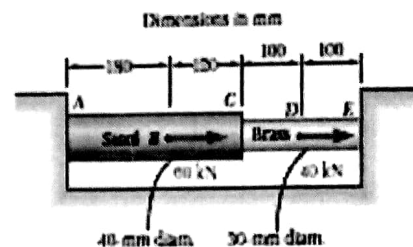
**Problem 1: (15 marks)**

Knowing that the allowable stress in section ABD is 10 ksi, determine the largest force  $P$  that can be applied to the bracket shown.E.



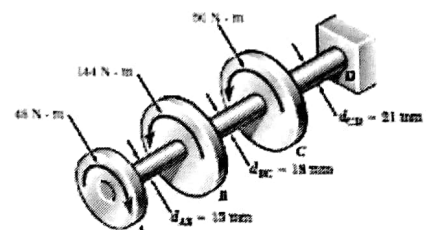
**Problem 2: (10 marks)**

Two cylindrical rods, one of steel and the other of brass, are joined at C and restrained by rigid supports at A and E. For the loading shown and knowing that  $E_s = 200$  GPa and  $E_b = 105$  GPa, determine: (a) the reactions at A and E, (b) the deflection of point C.



**Problem 3: (15 marks)**

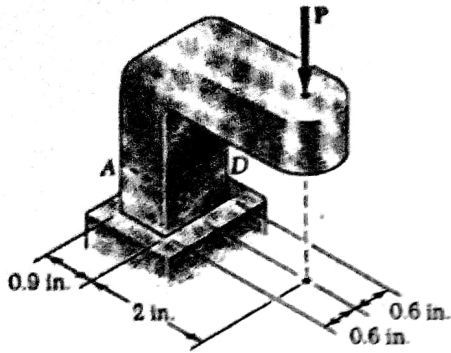
Knowing that each of the shafts AB, BC, and CD consists of a solid circular rod, determine (a) the shaft in which the maximum shearing stress occurs, (b) the magnitude of that stress.



Wish you the Best of Luck

Dr./ Mona Abou-Elaz

### Problem 4



4.109 Knowing that the allowable stress in section  $ABD$  is 10 ksi, determine the largest force  $P$  which can be applied to the bracket shown.

$$A = (1.2)(0.9) = 1.08 \text{ in}^2$$

$$I = \frac{1}{12}(1.2)(0.9)^3 = 72.9 \times 10^{-3} \text{ in}^4$$

$$c = \frac{1}{2}(0.9) = 0.45 \text{ in.}$$

$$e = 2 + 0.45 = 2.45 \text{ in.}$$

$$\sigma = \frac{P}{A} + \frac{Mc}{I} = \frac{P}{A} + \frac{Pec}{I} = PK$$

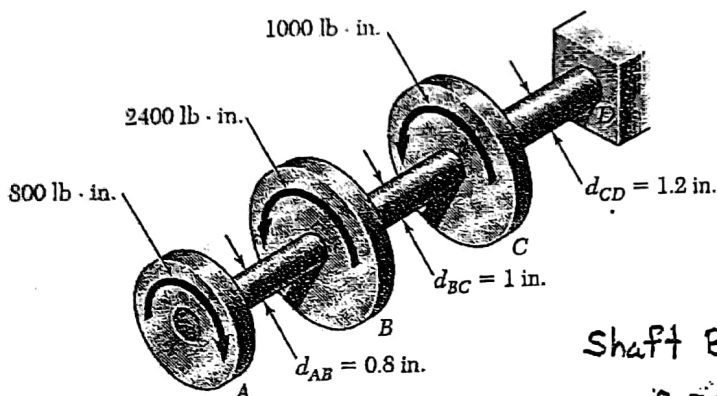
$$K = \frac{1}{A} + \frac{ec}{I} = \frac{1}{1.08} + \frac{(2.45)(0.45)}{72.9 \times 10^{-3}} = 16.049 \text{ in}^{-2}$$

$$P = \frac{\sigma}{K} = \frac{10}{16.049} = 0.623 \text{ kip}$$

$$P = 623 \text{ lb.}$$

### Problem 3

3.9 Knowing that each of the shafts  $AB$ ,  $BC$ , and  $CD$  consist of solid circular rods, determine (a) the shaft in which the maximum shearing stress occurs, (b) the magnitude of that stress.



$$\text{Shaft } AB: T = 800 \text{ lb}\cdot\text{in}$$

$$c = \frac{1}{2}d = 0.4 \text{ in.}$$

$$\tau_{max} = \frac{Tc}{J} = \frac{2T}{\pi c^3}$$

$$\tau_{max} = \frac{(2)(800)}{\pi (0.4)^3} = 7957 \text{ psi}$$

$$\text{Shaft } BC: T = -800 + 2400 = 1600 \text{ lb}\cdot\text{in}$$

$$c = \frac{1}{2}d = 0.5 \text{ in.}$$

$$\tau_{max} = \frac{(2)(1600)}{\pi (0.5)^3} = 8149 \text{ psi (largest)}$$

$$\text{Shaft } CD: T = -800 + 2400 + 1000 = 2600 \text{ lb}\cdot\text{in}$$

$$c = \frac{1}{2}d = 0.6 \text{ in.}$$

$$\tau_{max} = \frac{(2)(2600)}{\pi (0.6)^3} = 7663 \text{ psi}$$

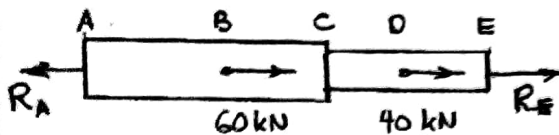
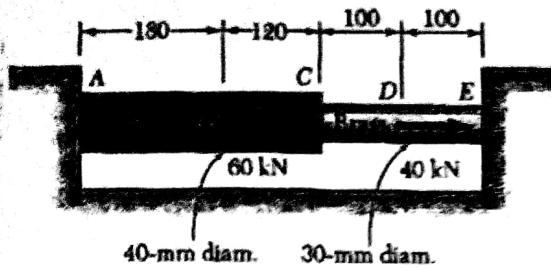
Answers: (a) Shaft BC

(b) 8.15 ksi

# Problem 2

2.39 Two cylindrical rods, one of steel and the other of brass, are joined at C and restrained by rigid supports at A and E. For the loading shown and knowing that  $E_s = 200$  GPa and  $E_b = 105$  GPa, determine (a) the reactions at A and E, (b) the deflection of point C.

Dimensions in mm



$$A \text{ to } C: E = 200 \times 10^9 \text{ Pa}$$

$$A = \frac{\pi}{4} (40)^2 = 1.25664 \times 10^3 \text{ mm}^2 = 1.25664 \times 10^{-3} \text{ m}^2$$

$$EA = 251.327 \times 10^6 \text{ N}$$

$$C \text{ to } E: E = 105 \times 10^9 \text{ Pa}$$

$$A = \frac{\pi}{4} (30)^2 = 706.86 \text{ mm}^2 = 706.86 \times 10^{-6} \text{ m}^2$$

$$EA = 74.220 \times 10^6 \text{ N}$$

$$A \text{ to } B: P = R_A$$

$$L = 180 \text{ mm} = 0.180 \text{ m}$$

$$\delta_{AB} = \frac{PL}{EA} = \frac{R_A(0.180)}{251.327 \times 10^6} = 716.20 \times 10^{-12} R_A$$

$$B \text{ to } C: P = R_A - 60 \times 10^3$$

$$L = 120 \text{ mm} = 0.120 \text{ m}$$

$$\delta_{BC} = \frac{PL}{EA} = \frac{(R_A - 60 \times 10^3)(0.120)}{251.327 \times 10^6} = 447.47 \times 10^{-12} R_A - 26.848 \times 10^{-6}$$

$$C \text{ to } D: P = R_A - 60 \times 10^3$$

$$L = 100 \text{ mm} = 0.100 \text{ m}$$

$$\delta_{CD} = \frac{PL}{EA} = \frac{(R_A - 60 \times 10^3)(0.100)}{74.220 \times 10^6} = 1.34735 \times 10^{-9} R_A - 80.841 \times 10^{-6}$$

$$D \text{ to } E: P = R_A - 100 \times 10^3$$

$$L = 100 \text{ mm} = 0.100 \text{ m}$$

$$\delta_{DE} = \frac{PL}{EA} = \frac{(R_A - 100 \times 10^3)(0.100)}{74.220 \times 10^6} = 1.34735 \times 10^{-9} R_A - 134.735 \times 10^{-6}$$

$$A \text{ to } E: \delta_{AE} = \delta_{AB} + \delta_{BC} + \delta_{CD} + \delta_{DE} = 3.85837 \times 10^{-9} R_A - 242.424 \times 10^{-6}$$

$$\text{Since point E cannot move relative to A, } \delta_{AE} = 0$$

$$(a) \quad 3.85837 \times 10^{-9} R_A - 242.424 \times 10^{-6} = 0 \quad R_A = 62.831 \times 10^3 \text{ N} \quad 62.8 \text{ kN} \leftarrow$$

$$R_E = R_A - 100 \times 10^3 = 62.8 \times 10^3 - 100 \times 10^3 = -37.2 \times 10^3 \text{ N} \quad 37.2 \text{ kN} \leftarrow$$

$$(b) \quad \delta_C = \delta_{AB} + \delta_{BC} = 1.16367 \times 10^{-9} R_A - 26.848 \times 10^{-6}$$

$$= (1.16369 \times 10^{-9})(62.831 \times 10^3) - 26.848 \times 10^{-6}$$

$$= 46.3 \times 10^{-6} \text{ m}$$

$$46.3 \mu\text{m} \rightarrow$$