



**QUESTION NO. One:**

**(a) Pick up the most appropriate statement of the multiple-choice answers .**

**1. Heat conducted through unit area and unit thick face per unit time when temperature difference between opposite faces is unity, is called**

- (a) thermal resistance                      (b) thermal diffusivity    (c) temperature gradient  
 (d) thermal conductivity                (e) heat-transfer.

**2. The value of the critical radius of insulation  $r_0$  for a cylinder is**

- (a)  $k/h$                       (b)  $2h/k$                       (c)  $2k/h$                       (d) None of these

**3. The ratio of heat flow  $Q_1/Q_2$  from two walls of same thickness having their thermal conductivities as  $k_1 = 2 k_2$  will be**

- (a) 1                      (b) 0.5                      (c) 2                      (d) 0.25                      (e) 4.

**4. Emissivity of a white polished body in comparison to a black body is**

- (a) higher                      (b) lower                      (c) same  
 (d) depends upon the shape of the body                      (e) none of the above.

**5. The heat dissipation from an finite fin and losing heat at the tip is given by:**

- (a)  $\sqrt{hPkA}(t_o - t_\infty)$                       (b)  $\sqrt{hPkA}(t_o - t_\infty) \tanh ml$   
 (c)  $\sqrt{hPkA}(t_o - t_\infty) \frac{\tanh ml + (h/mk)}{1 + (h/km)\tanh ml}$                       (d)  $\sqrt{hPkA}(t_o - t_\infty) \frac{\tanh ml + (h/k)m}{1 + (h/km)\tanh ml}$

**6. The vertical walls of a boiler furnace of size 4 m by 3 m by 3 m high. The walls are constructed from an inner fire brick wall 25 cm thick of thermal conductivity 0.4 W/m K, a layer of ceramic blanket insulation of thermal conductivity 0.2 W/m K and 8 cm thick, and a steel protective layer of thermal conductivity 55 W/m K and 2 mm thick. The inside temperature of the fire brick layer was measured at 600 °C and the temperature of the outside of the insulation 60 °C. The rate of heat loss may be :**

- (a)  $Q = 6320.96 \text{ W}$     (b)  $Q = 9320.96 \text{ W}$     (c)  $Q = 6020.96 \text{ W}$     (d)  $Q = 6300 \text{ W}$

- (b) A steam pipe with  $ID$  and  $OD$  as 100 and 170 mm is covered with two layers of insulation, 30 mm and 50 mm thick. Thermal conductivities of the insulating materials are 0.175 and 0.093 W/m K respectively while that of steel is 50 W/m K. The inner surface of the pipe is at 300 °C while the outer layer surface is at 50°C. Determine the heat loss from the pipe and the layer contact temperatures.
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### Question (2)

- a) Starting with energy balance on a rectangular volume element derive the Poisson Equation. (Use neat sketches)
- b) Atmospheric air at  $t_{\infty} = 40^{\circ}\text{C}$  with a free-stream velocity  $u_{\infty} = 8 \text{ m/s}$  flows along a flat plate  $L = 5 \text{ m}$  long and width of 2 m which is maintained at a uniform temperature of  $100^{\circ}\text{C}$ . The average drag coefficient  $C_m$  is  $2.13 \times 10^{-3}$ . Calculate the drag force over the entire length of the plate and the total heat transfer rate  $Q$  from the plate to the air .

*The physical properties of atmospheric air at  $t_f = (t_w + t_{\infty})/2 = (100 + 40)/2 = 70^{\circ}\text{C}$*

*are:*

$$k = 0.0295 \text{ W/(m.}^{\circ}\text{C)}, \nu = 2.005 \times 10^{-5} \text{ m}^2/\text{s}, Pr = 0.699$$

$$Nu_m = 0.036 Pr^{0.34} (Re_L^{0.8} - 9200)$$

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*Good Luck*  
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# Mid term Model Answer

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## Question No. one

(a)

(1)  $\longrightarrow$  (d)

(2)  $\longrightarrow$  (a)

(3)  $\longrightarrow$  (c)

(4)  $\longrightarrow$  (b)

(5)  $\longrightarrow$  (c)

(6)  $\longrightarrow$  (a)

b)

Given:  $r_1 = 50 \text{ mm}$

$r_2 = 85 \text{ mm}$

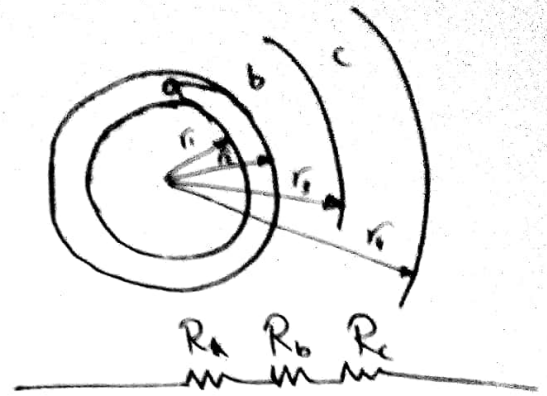
$r_3 = 115 \text{ mm}$

$r_4 = 165 \text{ mm}$

$k_a = 50 \text{ W/m.k}$

$k_b = 0.175 \text{ W/m.k}$

$k_c = 0.093 \text{ W/m.k}$



$t_1 = 300^\circ \text{C}$

$t_4 = 50^\circ \text{C}$

$$R_{\text{total}} = R_a + R_b + R_c$$

$$= \frac{\ln(85/50)}{2\pi \times 50} + \frac{\ln(115/85)}{2\pi \times 0.175} + \frac{\ln(165/115)}{2\pi \times 0.093}$$

$$= 1.69 \times 10^{-3} + 0.275 + 0.618 = 0.895$$

$$Q = \frac{t_1 - t_4}{R_{\text{total}}} = \frac{300 - 50}{0.895} = 279.4 \text{ W/m}$$

$$t_2 = t_1 - QR_a = 300 - 279.4 \times 1.69 \times 10^{-3} = 299.5^\circ \text{C}$$

$$t_3 = t_4 + QR_c = 50 + 279.4 \times 0.618 = 222.67^\circ \text{C}$$

## question (2)

a) as mentioned at the source.

b)

$$\rho = \frac{P}{RT} = \frac{101325}{287 \cdot 343} = 1.03 \text{ kg/m}^3$$

$$F = w L C_m \frac{\rho u^2}{2} \\ = 2 \cdot 5 \cdot 2.13 \cdot 10^{-3} \cdot \frac{1.03 \cdot 8^2}{2} = 0.702 \text{ N}$$

$$Re = \frac{uL}{\nu} = \frac{8 \cdot 5}{2.005 \cdot 10^{-5}} = 1995012.5$$

$$Nu = 0.036 \cdot 0.699^{0.34} (1995012.5^{0.8} - 9200) = 3201.2$$

$$h = \frac{Nu \cdot k}{L} = \frac{3201.2 \cdot 0.0295}{5} = 18.89 \text{ W/m}^2 \cdot \text{K}$$

$$Q = w L h (t_w - t_\infty)$$

$$= 2 \cdot 5 \cdot 18.89 (100 - 40) = 11332.4 \text{ W} \\ = 11.33 \text{ kW}$$